

ME 4555 - Lecture 12 - More Laplace Transforms

①

Recall: the Laplace transform is a technique/tool for solving the equations of motion that describe an LTI (linear time-invariant) system. These are linear ODEs.

Some examples

$$f(t) \xleftrightarrow{\mathcal{L}\{\cdot\}} F(s)$$

$$\xleftrightarrow{\mathcal{L}^{-1}\{\cdot\}} f(t)$$

definition:

$$F(s) \equiv \mathcal{L}\{f(t)\} = \int_0^{\infty} f(t) e^{-st} dt$$

$$H(t) \quad \frac{1}{s}$$

$$af(t) \quad aF(s)$$

$$f(t) + g(t) \quad F(s) + G(s)$$

} linearity

$$\frac{df(t)}{dt} \quad sF(s) - f(0)$$

$$\frac{d^2f(t)}{dt^2} \quad s^2F(s) - sf(0) - f'(0)$$

$$e^{-at} \quad \frac{1}{s+a}$$

$$\frac{1}{a}(1 - e^{-at}) \quad \frac{1}{s(s+a)}$$

System EOM
(ODEs)
in variable $x(t)$

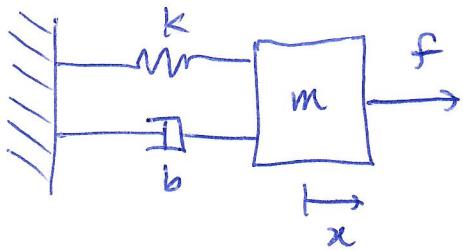
\mathcal{L}

Algebraic
Equations
in variable $X(s)$

Solve
for $X(s)$

$$\left(\begin{array}{l} \text{obtain solution} \\ x(t) = \dots \end{array} \right) \xleftarrow{\mathcal{L}^{-1}} \left(\begin{array}{l} \text{obtain solution} \\ X(s) = \dots \end{array} \right)$$

Ex : spring - mass - damper.



$$\text{EOM: } m\ddot{x} + b\dot{x} + kx = f$$

Take Laplace transform, $x(t) \xrightarrow{\mathcal{L}} X(s)$ and $f(t) \xrightarrow{\mathcal{L}} F(s)$.

$$m(s^2X - s\dot{x}(0) - \ddot{x}(0)) + b(sX - x(0)) + kx = F$$

$$\Rightarrow (ms^2 + bs + k)X = F + (ms + b)x(0) + m\dot{x}(0)$$

$$\Rightarrow X(s) = \left(\frac{1}{ms^2 + bs + k} \right) F(s) + \left(\frac{(ms+b)x(0) + m\dot{x}(0)}{ms^2 + bs + k} \right)$$

↑ ↑
 response input
 { transfer function input }
 particular solution
 (depends on input)

homogeneous solution, depends on initial conditions $x(0), \dot{x}(0)$ (initial position + velocity of mass).

the particular solution solves $m\ddot{x} + b\dot{x} + kx = f$ when $x(0) = \dot{x}(0) = 0$.

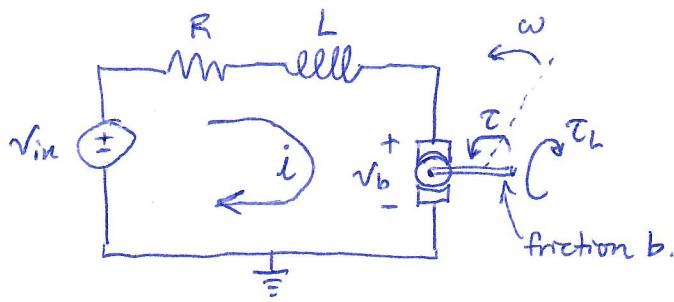
The homogeneous solution solves $m\ddot{x} + b\dot{x} + kx = 0$ for given $x(0)$, $\dot{x}(0)$.

The transfer function $G(s) = \frac{X(s)}{F(s)} = \frac{1}{ms^2 + bs + k}$ tells us how the system responds to input (when initial conditions are zero).

Ex: DC motor.

(3)

GOAL: Find $\omega(t)$ in terms of $V_{in}(t)$



time-domain

s-domain (take Laplace transform)

$$(1) \quad V_{in}(t) - V_b(t) = R i(t) + L \frac{di(t)}{dt} \rightarrow V_{in}(s) - V_b(s) = RI(s) + L(sI(s) - i(0))$$

$$(2) \quad T(t) = K_t i(t) \rightarrow T(s) = K_t I(s)$$

$$(3) \quad J\ddot{\omega}(t) = T(t) - b\omega(t) - T_L(t) \rightarrow J(s\Delta\omega(s) - \omega(0)) = T(s) - b\Delta\omega(s) - T_L(s)$$

$$(4) \quad V_b(t) = K_b \omega(t) \rightarrow V_b(s) = K_b \Delta\omega(s)$$

↓
sub (2) → (3)
and (4) → (1)

↓
set initial conditions to zero
 $i(0) = 0, \omega(0) = 0$.

$$(5) \quad V_{in}(t) = R i(t) + L \frac{di(t)}{dt} + K_b \omega(t)$$

$$\left\{ \begin{array}{l} V_{in} - V_b = (R + Ls) I \\ T = K_t I \end{array} \right.$$

$$(6) \quad -T_L(t) = J\dot{\omega}(t) + b\omega(t) - K_t i(t)$$

↓
solve (6) for $i(t)$

$$T - T_L = (b + Js) \Delta\omega$$

$$V_b = K_b \Delta\omega$$

$$(7) \quad i(t) = \frac{1}{K_t} [J\dot{\omega}(t) + b\omega(t) + T_L(t)]$$

(continued on next page.)

↓
substitute (7) into (5)

$$(8) \quad \left[\frac{LJ}{K_t} \ddot{\omega} + \left(\frac{RJ + Lb}{K_t} \right) \dot{\omega} + \left(\frac{Rb}{K_t} + K_b \right) \omega = V_{in} - \frac{L}{K_t} \dot{T}_L - \frac{R}{K_t} T_L \right]$$

(4)

$$\left. \begin{array}{l} (1) \quad V_m - V_b = (Ls + R) I \\ (2) \quad T = K_t I \\ (3) \quad T - T_L = (Js + b) \omega \\ (4) \quad V_b = K_b \omega \end{array} \right\} \begin{array}{l} \text{DC motor equations} \\ \text{in the } s\text{-domain.} \end{array}$$

inputs: $\{V_m(s), T_L(s)\}$, states: $\{I(s), \omega(s)\}$, intermediate quantities: $\{T(s), V_b(s), I(s)\}$

We want to relate $V_m(s)$ and $\omega(s)$ so we must eliminate $\{I(s), V_b(s), T(s)\}$.

substitute (2) \rightarrow (3) and (4) \rightarrow (1).

$$(5) \quad V_m = (Ls + R) I + K_b \omega$$

$$(6) \quad -T_L = (Js + b) \omega - K_t I$$

$$\text{Solve for } I \text{ in (6): } I = \left(\frac{Js + b}{K_t} \right) \omega + \frac{1}{K_t} T_L \quad (7).$$

Substitute this into (5):

$$(8) \quad V_m = (Ls + R) \left[\left(\frac{Js + b}{K_t} \right) \omega + \frac{1}{K_t} T_L \right] + K_b \omega.$$

Expand and rearrange: $\{\text{same as (8) on prev. page!}\}$

$$\underbrace{\left(\frac{LJ}{K_t} s^2 + \left(\frac{RJ + Lb}{K_t} \right) s + \left(\frac{Rb}{K_t} + K_b \right) \right)}_{H(s)} \omega = V_m - \left(\frac{L}{K_t} s + \frac{R}{K_t} \right) T_L$$

Transfer function from $V_m \rightarrow \omega$: $\frac{\omega}{V_m} = \frac{1}{H(s)}$ $\{\text{what you get if } T_L = 0\}$
 $\{\text{and } V_m \text{ is only input.}\}$

Transfer function from $T_L \rightarrow \omega$: $\frac{\omega}{T_L} = \frac{Ls + R}{K_t H(s)}$ $\{\text{what you get if } V_m = 0\}$
 $\{\text{and } T_L \text{ is only input.}\}$

Ex A general linear ODE. Input $u(t)$, output $y(t)$. (5)

$$y^{(n)}(t) + a_{n-1}y^{(n-1)}(t) + \dots + a_1y(t) + a_0y(t) = b_{n-1}u^{(n-1)}(t) + \dots + b_1u(t) + b_0u(t)$$

↓ take Laplace transform with zero initial conditions.

$$(s^n + a_{n-1}s^{n-1} + \dots + a_1s + a_0)Y(s) = (b_{n-1}s^{n-1} + \dots + b_1s + b_0)U(s)$$

↓ rearrange.

Transfer function:

$$G(s) = \frac{Y(s)}{U(s)} = \frac{b_{n-1}s^{n-1} + \dots + b_1s + b_0}{s^n + a_{n-1}s^{n-1} + \dots + a_1s + a_0}$$

- ★ The transfer function for an LTI system is always a rational function of s (ratio of two polynomials)
- ★ the degree of the denominator (n) is called the order of the system. So $G(s)$ above is an n^{th} order system.
- ★ The order of a system is equal to the number of integrators needed to represent it in simulink. i.e. a spring-mass-damper system is a 2nd order system because its transfer function is $\frac{1}{ms^2 + bs + k}$ and the denominator has degree 2.